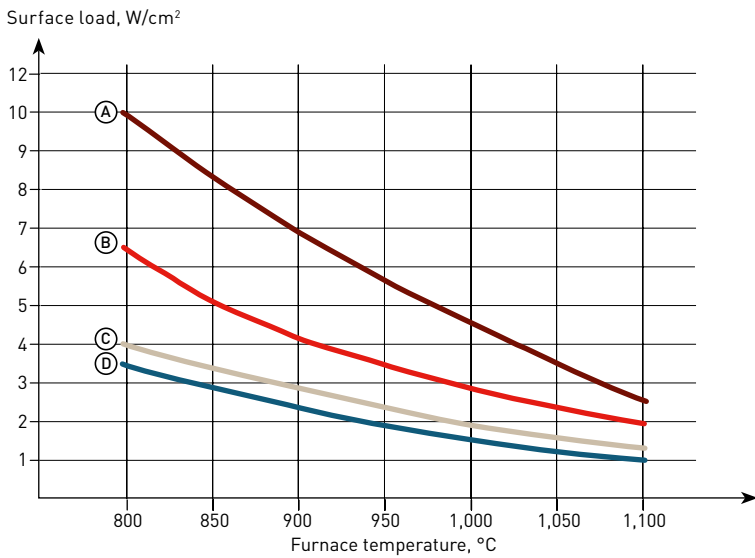


MAXIMUM RECOMMENDED SURFACE LOADS FOR NIKROTHAL® ALLOYS IN INDUSTRIAL FURNACES



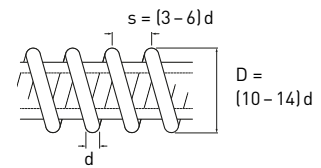
(A) Freely radiating corrugated wire elements



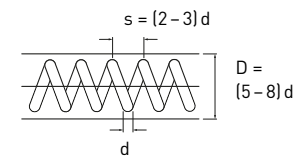
(B) Freely radiating corrugated strip elements



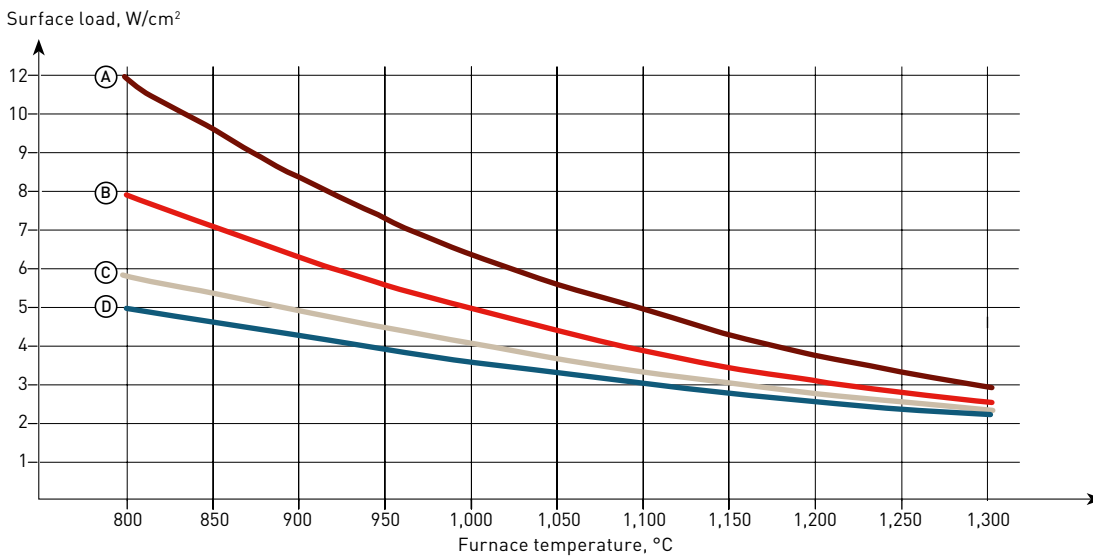
(C) Spiral elements on ceramic tubes



(D) Heating elements in grooves



MAXIMUM RECOMMENDED SURFACE LOADS FOR KANTHAL® A-1, KANTHAL® AF AND KANTHAL® APM ALLOYS IN INDUSTRIAL FURNACES



Note: The diagrams are valid for thyristor control. For on-off control lower surface loads should be chosen (about - 20%).

DESIGN OF WIRE ELEMENTS

CALCULATION OF WIRE DIAMETER

DIRECT CALCULATION METHOD

The diameter that will result in the desired surface load, given the electrical input data, can be calculated as:

$$d = \frac{1}{k_d} \sqrt[3]{I^2 \frac{\rho C_t}{p_s}}$$

where:

- I = Current
- ρ = Resistivity
- C_t = Temperature factor
- p_s = Surface load of heating element
- k_d = Wire diameter form factor

For metric units, with ρ in $\Omega \text{ mm}^2/\text{m}$ and p_s in W/cm^2 , $k_d = 2.91$, and d will be in mm.

For imperial units, with ρ in $\Omega \text{ circ. mil}/\text{ft}$ and p_s in W/in^2 , $k_d = 335$, and d will be in inch.

Example:

- Power, $W = 20 \text{ kW}$
- Voltage, $U = 220 \text{ V}$
- Target surface load, $p_s = 4.0 \text{ W}/\text{cm}^2$ [26 W/in^2]
- Wire temperature = $1,200^\circ\text{C}$ [2,190°F]
- Material = Kanthal® AF
- Resistivity, $\rho = 1.39 \Omega \text{ mm}^2/\text{m}$ [836 $\Omega \text{ circ. mil}/\text{ft}$]
- Temperature factor, C_t ($1,200^\circ\text{C}$) = 1.06

First determine the current, I:

$$I = \frac{P}{U} = \frac{20,000}{220} = 90.9 \text{ A}$$

The diameter can then be calculated:

$$d = \frac{1}{2.91} \sqrt[3]{90.9^2 \frac{1.39 \times 1.06}{4.0}} = 4.98 \text{ mm (0.196 in)}$$

The nearest standard size is 5.0 mm (0.197 in). The actual surface load at this power and voltage will then be $3.95 \text{ W}/\text{cm}^2$ [25.5 W/in^2].

TABLE LOOKUP METHOD

Wire diameter can alternatively be chosen using the ratio η , i.e. surface area, A_c , to cold resistance, R_{20} . The ratio is presented as in the tables section, page 73.

Given the electrical data, element temperature, and target surface load, η can be calculated as:

$$\eta = \frac{A_c}{R_{20}} = I^2 \frac{C_t}{p_s}$$

where:

- A_c = Surface area of the conducting wire
- R_{20} = Cold resistance
- I = Current
- C_t = Temperature factor
- p_s = Surface load of heating element

Having determined the target η , a suitable wire diameter can be selected from the tables. Whether this size actually suits the element concerned should be considered in relation to the operating conditions.

Example:

$$\eta = \frac{90.9^2 \times 1.06}{4.0} = 2,194 \frac{\text{cm}^2}{\Omega} \left(340 \frac{\text{in}^2}{\Omega} \right)$$

Under the η column in the table, the nearest value is $2,220 \text{ cm}^2/\Omega$ [344 in^2/Ω], giving a diameter of 5.0 mm (0.197 in). The actual surface load at this power and voltage will then be $3.95 \text{ W}/\text{cm}^2$ [25.5 W/in^2].

CALCULATION OF WIRE LENGTH

Having determined wire type and diameter, the next stage in arriving at a spiral element is to calculate wire length. The first step is to determine the cold resistance, R_{20} , of the selected wire:

$$R_{20} = \frac{R_T}{C_t} = \frac{U^2}{PC_t}$$

where:

- R_{20} = Cold resistance
- R_T = Hot resistance
- C_t = Temperature coefficient
- U = Voltage
- P = Power

The resistance per unit length, $R_{20/m}$, can be calculated as:

$$R_{20/m} = \frac{4\rho}{\pi d^2}$$

Alternatively, the resistance per meter (or per ft) has been precalculated for material and diameter combinations and can be found in the tables. From this, the wire length, ℓ , can be calculated as:

$$\ell = \frac{R_{20}}{R_{20/m}}$$

Example:

- Wire diameter, $d = 5.0$ mm (0.197 in)
- Power, $P = 20$ kW
- Voltage, $U = 220$ V
- Wire temperature = $1,200^\circ\text{C}$ ($2,190^\circ\text{F}$)
- Material = Kanthal[®] AF
- Resistivity, $\rho = 1.39 \Omega \text{ mm}^2/\text{m}$ ($836 \Omega \text{ circ. mil}/\text{ft}$)
- Temperature factor, C_t ($1,200^\circ\text{C}$) = 1.06

$$R_T = \frac{U^2}{P} = \frac{220^2}{20,000} = 2.42 \Omega$$

$$R_{20} = \frac{R_T}{C_t} = \frac{2.42}{1.06} = 2.28 \Omega$$

The resistance per unit length, $R_{20/m}$ is:

$$R_{20/m} = \frac{4 \times 1.39}{3.14 \times 5.0^2} = 0.0708 \frac{\Omega}{\text{m}} \left(0.0216 \frac{\Omega}{\text{ft}} \right)$$

The total wire length is:

$$\ell = \frac{2.28}{0.0708} = 32.2 \text{ m (106 ft)}$$

COIL ELEMENT DIMENSIONS

Having determined the wire diameter and length, the next step is to select the external diameter of the coil. Regarding values for the ratio of coil external diameter, D to wire diameter, d , see page 51.

Smaller ratios cause too great a winding strain on the wire; larger ratios produce a weaker, more flimsy coil.

For a given wire length, ℓ , the number of turns, w is:

$$w = \frac{\ell}{\pi(D - d)}$$

where the length should be converted to the same unit as diameter, e.g. from m to mm (multiply ℓ by 1,000) or ft to in (multiply ℓ by 12).

The close-wound coil length, L_w , is:

$$L_w = wd$$

Recommended values for the pitch, s , see page 51.

Stretched coil length can finally be calculated as:

$$L = \frac{s}{d} L_w$$

Example:

- Wire diameter, $d = 5.0$ mm (0.197 in)
- Wire length, $\ell = 32.2$ m (106 ft)
- Coil outer diameter, $D = 27.5$ mm (1.08 in)
- Pitch, $s =$ between 10 mm (0.394 in) and 20 mm (0.787 in)

$$w = \frac{32,200}{3.14 \times (27.5 - 5.0)} \approx 456$$

Close-wound coil length,

$$L_w = 456 \times 5.0 = 2,280 \text{ mm (89.8 in)}$$

Stretched length,

$$\begin{aligned} \text{min: } L &= 2 \times 2,280 = 4,560 \text{ mm (180 in)} \\ \text{max: } L &= 4 \times 2,280 = 9,120 \text{ mm (359 in)} \end{aligned}$$

DESIGN OF CORRUGATED STRIP ELEMENTS

In designing corrugated strip elements, the procedure is first to determine the width, thickness, and length of the strip and then the corrugation parameters: pitch, depth (or height) of corrugation, etc. Instructions are given for both parallel and stretched corrugated elements.

Strip size depends on operating conditions centered around surface loading and on the type of alloy selected. In turn, selection of an alloy is based on the same parameters plus furnace atmosphere, heating/cooling cycles, and element temperature. For recommended values of surface load, see graphs in page 51.

Example:

Power, $W = 12 \text{ kW}$
 Voltage, $U = 110 \text{ V}$
 Target surface load, $p_s = 1.0 \text{ W/cm}^2 \text{ (6.45 W/in}^2\text{)}$
 Material = Kanthal® A-1
 Resistivity, $\rho = 1.45 \text{ } \Omega \text{ mm}^2/\text{m (872 } \Omega \text{ circ. mil/ft)}$
 Temperature factor, $C_t \text{ (1,200}^\circ\text{C)} = 1.04$

Table lookup method

An alternative method of calculating the strip section is calculate the surface area to cold resistance ratio and find a suitable strip dimension in the tables:

$$\eta = \frac{A_c}{R_{20}} = I^2 \frac{C_t}{p_s}$$

Example

$$\eta = \frac{109.1^2 \times 1.04}{1.0} = 12,400 \frac{\text{cm}^2}{\Omega} \left(1,920 \frac{\text{in}^2}{\Omega} \right)$$

The nearest standard size is $2.0 \times 20 \text{ mm (0.079} \times 0.79 \text{ in)}$ with $\eta = 12,100 \text{ cm}^2/\Omega \text{ (1875 in}^2/\Omega\text{)}$. The actual surface load at this power and voltage will then be $1.02 \text{ W/cm}^2 \text{ (6.6 W/in}^2\text{)}$. Note that a $0.7 \times 35 \text{ mm (0.028} \times 1.4 \text{ in)}$ strip also has $\eta = 12,100 \text{ cm}^2/\Omega \text{ (1875 in}^2/\Omega\text{)}$, but is much less appropriate since the thickness ideally should be at least 1.5 mm (0.06 in) .

CALCULATION OF STRIP LENGTH

Having determined the strip width and thickness, the strip length, l , must be calculated. This requires an initial calculation of cold resistance, R_{20} :

$$R_{20} = \frac{R_T}{C_t} = \frac{U^2}{PC_t}$$

The resistance per unit length, $R_{20/m}$, can be calculated as:

$$\ell = \frac{R_{20}}{R_{20/m}}$$

Alternatively, the resistance per meter (or per ft) has been precalculated for material and diameter combinations and can be found in the tables. From this, the strip length, ℓ , can be calculated as:

$$R_{20/m} = \frac{\rho}{tb}$$

Example:

Strip dimensions, $t \times b = 2.0 \times 20 \text{ mm (0.079} \times 0.79 \text{ in)}$
 Power, $P = 12 \text{ kW}$
 Voltage, $U = 110 \text{ V}$
 Material = Kanthal® A-1
 Resistivity, $\rho = 1.45 \text{ } \Omega \text{ mm}^2/\text{m (872 } \Omega \text{ circ. mil/ft)}$
 Temperature factor, $C_t \text{ (1,200}^\circ\text{C)} = 1.04$

$$R_T = \frac{U^2}{P} = \frac{110^2}{12,000} = 1.01 \Omega$$

$$R_{20} = \frac{R_T}{C_t} = \frac{1.01}{1.04} = 0.97 \Omega$$

The resistance per unit length, $R_{20/m}$ is:

$$R_{20/m} = \frac{1.39}{2 \times 20} = 0.0363 \frac{\Omega}{m} \left(0.0110 \frac{\Omega}{ft} \right)$$

The total wire length is:

$$\ell = \frac{0.97}{0.0363} = 26.7 \text{ m (88 ft)}$$

CORRUGATED ELEMENT DIMENSIONS

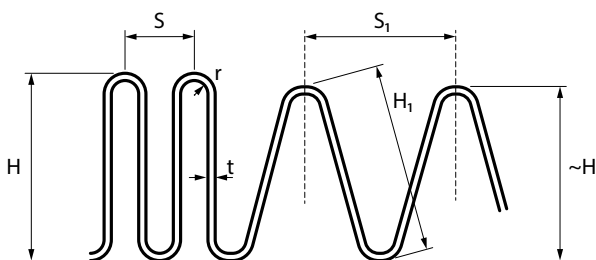
A corrugated element, with a parallel section and a stretched section, is shown in Fig. 1. As illustrated, the key design parameters for the element are the element (or corrugation) height, H , pitch, s , and bending radius, r . Other values necessary for element calculations are: Total element length, L_e (given by the furnace design), and strip thickness, t , width, b , and length, ℓ .

Parallel Corrugated Element

When the loops are parallel, there is a geometrical relationship between r and s :

$$r = \frac{s - 2t}{4}; \quad s = 4r + 2t$$

FIGURE 1



The element length, L_e , after parallel corrugation is then:

$$L_e = \frac{\ell(2r + t)}{H + 1.14r - 0.43t}$$

Alternatively, if the element length is pre-determined, the element height can be calculated as:

$$H = \frac{\ell}{L_e} (2r + t) - 1.14r + 0.43t$$

Stretched Elements

Stretching a corrugated element so that the loops are no longer parallel reduces the danger of deformation. A stretched element may be calculated by the same method as a parallel element. That is: the total (stretched) element length, L_1 , is given by the furnace design; t , b , and ℓ are determined by calculation. Element height, H , must be selected.

The pitch after stretching, s_1 , (center-to-center distance between hooks for hanging elements) is:

$$s_1 = \frac{sL_1}{L}$$

The element height can be calculated by:

$$H = \frac{\ell}{L_e} (2r + t) - 1.14r + 0.43t$$

The height of a stretched element, H , will be somewhat lower than that with parallel loops.

$$H \approx \sqrt{H_1 - \left(\frac{s_1}{2}\right)^2}$$

The difference in height will in most cases be small and may be neglected.

Example:

Strip dimensions, $t \times b = 2.0 \times 20 \text{ mm}$ (0.079 \times 0.79 in)
 Strip length: 26.7 m (88 ft)
 Element length, L_e : 3.00 m (9.84 ft)
 Ceramic support diameter, d_{sup} : 28 mm (1.1 in)
 Ceramic support max permissible width: 33 mm (1.3 in)

A bending radius, r , of at least 14 mm (0.55 in) is needed for the strip to fit around the $\varnothing 28 \text{ mm}$ ($\varnothing 1.1 \text{ in}$) ceramic supports. A slight clearance of 0.5 mm (0.02 in) should be added in addition to this.

$$r = \frac{d_{sup}}{2} + 0.5 = \frac{28}{2} + 0.5 = 14.5 \text{ mm (0.57 in)}$$

For parallel loops, the pitch, s , will then be:

$$s = 4r + 2t = 62 \text{ mm (2.44 in)}$$

The element height is then:

$$H = \frac{26.7}{3.00} (2 \times 14.5 + 2) - 1.14 \times 14.5 + 0.43 \times 2 = 260 \text{ mm (10.2 in)}$$

Check the dimensions:

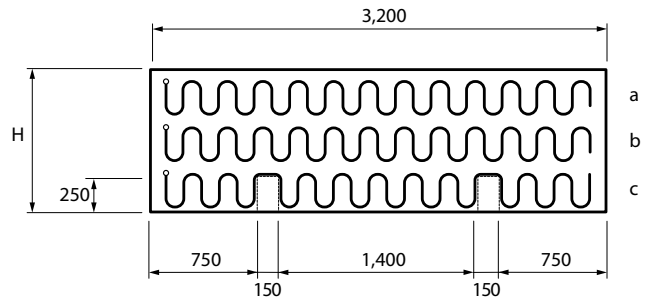
- $t = 2 \text{ mm}$ (0.079 in)
- $b = 20 \text{ mm}$ (0.79 in) = 10 t
- $s = 62 \text{ mm}$ (2.44 in) = 3.1 b
- $r = 14.5 \text{ mm}$ (0.57 in) = 7.25 t
- $H = 260 \text{ mm}$ (10.2 in)
- >1.5 mm (>0.06 in)
- [8–12] t
- $\leq 33 \text{ mm}$ (1.3 in)
- min. 50 mm (2.0 in)
- min. (4–5) t
- max. 150–400 mm (5.9–15.8 in)

All dimensions are within the recommended limits.

DESIGN OF FREELY RADIATING STRIP AND WIRE ELEMENTS

In order to show the element calculation methods for freely radiating wire and strip elements, a 200 kW fiber-lined furnace is chosen as a calculation example (see Fig. 2). The furnace is equipped with deep-corrugated elements mounted on ceramic hangers. In the first case, wire elements are used, and in the second case, strip elements. A comparison has been made at the end with strip elements of NiCr elements to show weight and cost savings by using Kanthal® AF.

FIGURE 2



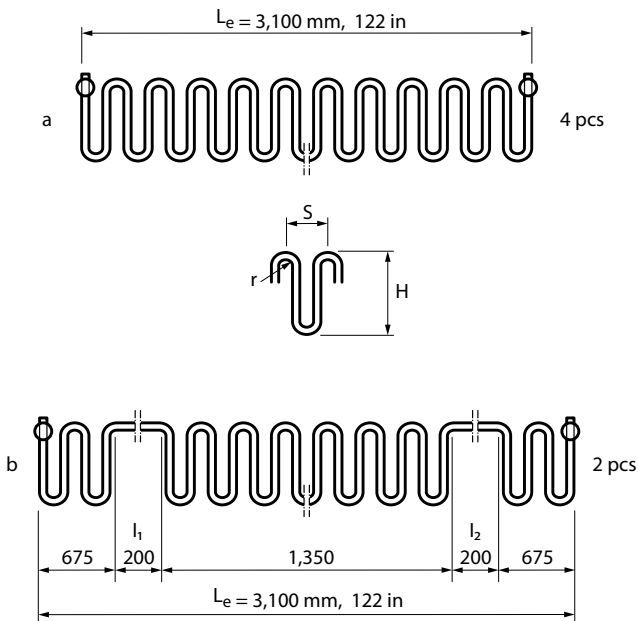
DEEP-CORRUGATED KANTHAL® AF WIRE ELEMENT

The element design is shown in Fig. 3. The furnace is equipped with four elements of type "a" and two elements of type "b".

FURNACE DATA

Power (total): 200 kW
 Furnace temperature, T_f : 1,100°C (2,010°F)
 Assumed element temperature, T_e (for calculations): 1,200°C (2,190 °F)
 Number of 3-phase groups: 1
 Power per phase: 66.67 kW
 Voltage: 380 V
 Current: 175.4 A
 Resistance: 2.166 Ω
 Number of elements in series: 2
 Element length, L_e : 3.1 m (10.2 ft)

FIGURE 3



DATA PER ELEMENT

Power, P: 33.33 kW
 Voltage, U: 190 V
 Current, I: 175.4 A
 Max surface load, p_s : 6.0 W/cm² (38.7 W/in²)
 Resistance, hot, R_T : 1.083 Ω
 Material: Kanthal® AF
 Resistivity, ρ : 1.39 Ω mm²/m
 Temperature factor, C_t (1,200°C): 1.06
 Terminals: Round, ℓ_u , length 350 mm (13.8 in)

CALCULATION OF TERMINALS

Determine the appropriate terminal diameter to handle the current and maintain the lowest terminal temperature.

With the terminal length, ℓ_u , and diameter, d_u , decided, the resistance of the two terminals, R_u , (one per end) can be calculated:

$$R_u = 2 \times \frac{4\ell_u\rho}{\pi d^2}$$

$$R_u = 2 \times \frac{4 \times 0.35 \times 1.39}{3.14 \times 16^2} = 0.0048 \Omega$$

CALCULATION OF WIRE DIAMETER

The preliminary wire diameter in mm (or in) is calculated as:

$$d = \frac{1}{k_d} \sqrt[3]{I^2 \frac{\rho C_t}{p_s}}$$

where k_d is 2.91 for metric units (or 335 for imperial). The diameter that would give a surface load of 6.0 W/cm² (38.7 W/in²) at full power is:

$$d = \frac{1}{2.91} \sqrt[3]{175.4^2 \frac{1.39 \times 1.06}{6.0}} = 6.74 \text{ mm (0.265 in)}$$

The closest standard dimension is 7.0 mm (0.276 in).

CALCULATION OF WIRE LENGTH

The cold resistance of the element, R_{20} , can be calculated as:

$$R_{20} = \frac{R_T}{C_t} - R_u$$

where R_u is the cold resistance of the two terminals combined.

$$R_{20} = \frac{1.083}{1.06} - 0.0048 = 1.017 \Omega$$

The wire length, ℓ , can then be calculated as:

$$\ell = \frac{\pi d^2 R_{20}}{4\rho}$$

$$\ell = \frac{3.14 \times 7.0^2 \times 1.017}{4 \times 1.39} = 28.1 \text{ m (92.2 ft)}$$

Based on this diameter and length, and that the density of Kanthal® AF is 7.15 kg/cm³ (0.258 lb/in³), the wire weight will be 7.7 kg (17 lb).

SURFACE LOAD

The surface load can be calculated using:

$$p_s = \frac{I^2 C_t}{\eta}$$

where η for the selected wire dimension can be found in the table. For $\varnothing 7.0$ mm Kanthal® AF it is 6,090 cm²/Ω.

$$p_s = \frac{175.4^2 \times 1.06}{6,090} = 5.35 \frac{\text{W}}{\text{cm}^2} \left(34.5 \frac{\text{W}}{\text{in}^2} \right)$$

An alternative is to use the formula:

$$p_s = \frac{I^2 \rho C_t}{24.67 \times d^3} \text{ (metric)}$$

$$p_s = \frac{I^2 \rho C_t}{3.77 \times 10^7 \times d^3} \text{ (imperial)}$$

BENDING RADIUS

A bending radius, r , of 9 mm (0.35 in) will be used in this example.

Maximum corrugation height

Three elements are suspended on a wall having a height of 1,000 mm (1,000/3 ≈ 333 mm per element [13.1 in/element]). Take into account that about 25% of this height per element should be reserved for clearance, leaving 75% of that height for the element.

$$H_{\max} = 333 \times 0.75 = 250 \text{ mm (9.84 in)}$$

This number can be used as a preliminary height when calculating the number of pitches.

CALCULATION OF NUMBER OF PITCHES

The number of pitches is calculated as:

$$N = \frac{0.5 \times [\ell - (\ell_1 + \ell_2 + \dots + \ell_n)]}{H + 1.14r - 0.43d}$$

where $\ell_1, \ell_2, \dots, \ell_n$ are straight sections of the element.

Element type "a" [Fig. 3], no straight sections:

$$N = \frac{0.5 \times 28,100}{250 + 1.14 \times 9.0 - 0.43 \times 7.0} = 54.6 \approx 55$$

Since the maximum height was used in this calculation, the result must be rounded up to 55. With 55 pitches, the average pitch, s , will be:

$$s = \frac{L_e}{N} = \frac{3,100}{55} = 56 \text{ mm (2.2 in)}$$

Check that $s \geq 4r + 2d$

$$4r + 2d = 4 \times 9.0 + 2 \times 7.0 = 50 \text{ mm (2.0 in)} \rightarrow \text{ok}$$

Element "b" [Fig. 3], two 200 mm long straight sections:

$$N = \frac{0.5 \times [28,100 - (200 + 200)]}{250 + 1.14 \times 9.0 - 0.43 \times 7.0} = 53.8 \approx 54$$

Rounding up, this gives 54 pitches. These 54 pitches need to be distributed over a total span of 2,700 mm (106.3 in), consisting of two 675 mm (26.6 in) wide outer parts, and the 1,350 mm (53.1 in) wide center part

between the supporting beams (see Fig. 3). The outer parts correspond to $\frac{1}{4}$ of the span each and should thus contain approximately $\frac{1}{4}$ of the 54 pitches, and in the same way since the center part corresponds to $\frac{1}{2}$ of the span, it should also have roughly half of the pitches. The exact numbers would be 13.5 pitches in the outer parts, and 27 pitches in the center. For a design using full pitches, the distribution will be 14 pitches in the outer parts and 26 in the center. The average pitch in the outer parts, s_1 , will thus be:

$$s_1 = \frac{675}{14} = 48 \text{ mm (1.9 in)}$$

and in the center:

$$s_2 = \frac{1,350}{26} = 52 \text{ mm (2.0 in)}$$

Check that $s \geq 4r + 2d$

$$4r + 2d = 4 \times 9.0 + 2 \times 7.0 = 50 \text{ mm (2.0 in)} \rightarrow \text{not ok}$$

Note that the pitch in the outer parts will be less than 50 mm. As an alternative solution the furnace design can be revised by moving the 200 mm [7.0 in] ceramic supports inwards by 25 mm [1.0 in] (see Fig. 4). The average pitch in the outer parts then become:

$$s_1 = \frac{700}{14} = 50 \text{ mm (2.0 in)}$$

and in the center:

$$s_2 = \frac{1,300}{26} = 50 \text{ mm (2.0 in)}$$

Check that $s \geq 4r + 2d$

$$4r + 2d = 4 \times 9.0 + 2 \times 7.0 = 50 \text{ mm (2.0 in)} \rightarrow \text{ok}$$

CALCULATION OF CORRUGATION HEIGHT

The element height, or corrugation height, is calculated as:

$$H = \frac{0.5 \times [\ell - (\ell_1 + \ell_2 + \dots + \ell_n)]}{N} - 1.14r + 0.43d$$

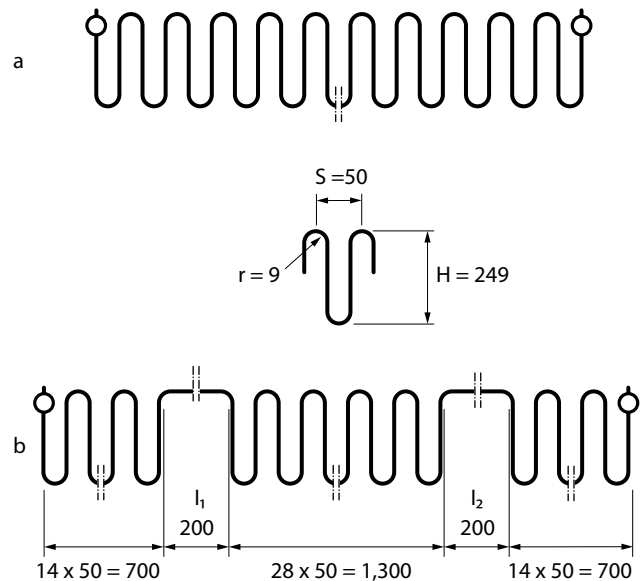
Element type "a" (Fig. 3), no straight sections:

$$H = \frac{0.5 \times (23,100)}{55} - 1.14 \times 9.0 + 0.43 \times 7.0 = 248 \text{ mm (9.8 in)}$$

Element "b" (Fig. 3), two 200 mm long straight sections:

$$H = \frac{0.5 \times [23,100 - (200 + 200)]}{54} - 1.14 \times 14.5 + 0.43 \times 2.5 = 249 \text{ mm (9.8 in)}$$

FIGURE 4



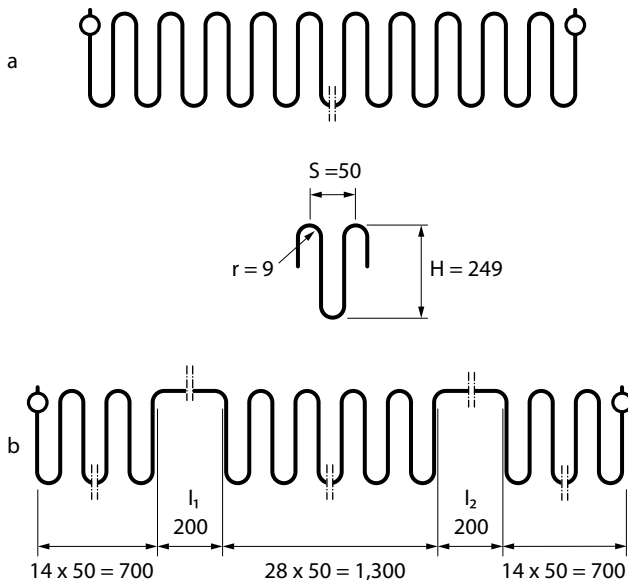
DEEP-CORRUGATED KANTHAL® AF STRIP ELEMENTS

The element design is shown in Fig. 5. The furnace is equipped with four elements of type "a" and two elements of type "b". As can be seen from the sketch, element "b" has two straight parts because of the supporting beams for the charge.

FURNACE DATA

Power (total): 200 kW
 Furnace temperature, T_f : 1,100°C (2,010°F)
 Assumed element temperature, T_e (for calculations): 1,200 °C (2,190 °F)
 Number of 3-phase groups: 1
 Power per phase: 66.67 kW
 Voltage: 220 V
 Current: 303 A
 Resistance, hot: 0.726 Ω
 Number of elements in series: 2
 Element length, L_e : 3.1 m (10.2 ft)

FIGURE 5



DATA PER ELEMENT

Power, P: 33.33 kW
 Voltage, U: 110 V
 Current, I: 303 A
 Max surface load, p_s : 3.0 W/cm² (19.4 W/in²)
 Resistance, hot, R_r : 0.363 Ω
 Material: Kanthal® AF
 Resistivity, ρ : 1.39 Ω mm²/m
 Temperature factor, Ct (1,200°C): 1.06
 Terminals: Round, length, ℓ_u : 350 mm (13.8 in)

CALCULATION OF TERMINALS

Determine the appropriate terminal diameter to handle the current and maintain the lowest terminal temperature.

With the terminal length, ℓ_u , and diameter, d_u , decided, the resistance of the two terminals, R_u , (one per end) can be calculated:

$$R_u = 2 \times \frac{4\ell_u\rho}{\pi d^2}$$

$$R_u = 2 \times \frac{4 \times 0.35 \times 1.39}{3.14 \times 20^2} = 0.0031 \Omega$$

CALCULATION OF STRIP SIZE

The preliminary strip thickness is calculated as:

$$t = k_t \times \sqrt[3]{I^2 \frac{\rho \times C_t}{p_s}}, k_t = \sqrt[3]{\frac{1}{20n(1+n)}}$$

where n is the desired ratio of width/thickness. The version of k_t above applies for the use of metric units. Aiming for a surface load of 3.0 W/cm² (19.4 W/in²), and n = 12 gives:

$$t = \sqrt[3]{\frac{1}{20 \times 12 \times (1 + 12)}} \times \sqrt[3]{303^2 \frac{1.39 \times 1.06}{3.0}} =$$

2.44 mm (0.096 in)

The closest standard dimension is 2.5 mm thickness (0.10 in). Aiming for a 12 times wide strip means that

$$b = 12 \times 2.5 = 30 \text{ mm (1.2 in)}$$

CALCULATION OF STRIP LENGTH

The cold resistance of the element, R_{20} , can be calculated as:

$$R_{20} = \frac{R_T}{C_t} - R_u$$

where R_u is the cold resistance of the two terminals combined.

$$R_{20} = \frac{0.363}{1.06} - 0.0031 = 0.339 \Omega$$

The strip length, ℓ , can then be calculated as:

$$\ell = \frac{R_{20}tb}{\rho}$$

$$\ell = \frac{0.339 \times 2.5 \times 30}{1.39} = 18.3 \text{ m (60 ft)}$$

Based on this width, thickness and length, and that the density of Kanthal® AF is 7.15 kg/cm³ [0.258 lb/in³], the strip weight will be 9.8 kg [21.6 lb].

SURFACE LOAD

The surface load can be calculated using:

$$p_s = \frac{I^2 C_t}{\eta}$$

where n for the selected strip dimension can be found in the table. For 2.5 × 30 mm Kanthal® AF it is 35,100 cm²/Ω.

$$p_s = \frac{303^2 \times 1.06}{35,100} = 2.77 \frac{W}{cm^2} \left(17.9 \frac{W}{in^2} \right)$$

An alternative is to use the formula:

$$p_s = \frac{1}{20n(1+n)} I^2 \frac{\rho C_t}{t^3} \text{ (metric)}$$

$$p_s = \frac{\pi}{96 \times 10^6 \times n(1+n)} I^2 \frac{\rho C_t}{t^3} \text{ (imperial)}$$

which gives the same result.

BENDING RADIUS

Ceramic supports to be used have a diameter of 28 mm (1.1 in) and permit a maximum strip width of 33 mm (1.3 in). This gives a minimum bending radius, r , of 14 mm (0.55 in). Add 0.5 mm (0.02 in) for clearance, giving:

$$r = \frac{28}{2} + 0.5 = 14.5 \text{ mm (0.57 in)}$$

MAXIMUM CORRUGATION HEIGHT

Three elements are suspended on a wall having a height of 1,000 mm [1,000/3 ≈ 333 mm per element [13.1 in/element]]. Take into account that about 25% of this height per element should be reserved for clearance, leaving 75% of that height for the element.

$$H_{max} = 333 \times 0.75 = 250 \text{ mm (9.84 in)}$$

This number can be used as a preliminary height when calculating the number of pitches.

CALCULATION OF NUMBER OF PITCHES

The number of pitches is calculated as:

$$N = \frac{0.5 \times [\ell - (\ell_1 + \ell_2 + \dots + \ell_n)]}{H + 1.14r - 0.43t}$$

where $\ell_1, \ell_2, \dots, \ell_n$, are the lengths of straight sections of the element.

Element type "a" (Fig. 5), no straight sections:

$$N = \frac{0.5 \times 18,300}{250 + 1.14 \times 14.5 - 0.43 \times 2.5} = 34.45 \approx 35$$

Since the maximum height was used in this calculation, the result must be rounded up to 35. With 35 pitches, the average pitch, s , will be:

$$s = \frac{L_e}{N} = \frac{3,100}{35} = 89 \text{ mm (3.5 in)}$$

Check that $s \geq 4r + 2t$:

$$4r + 2t = 4 \times 14.5 + 2 \times 2.5 = 63 \text{ mm (2.5 in)} \rightarrow \text{ok}$$

Element "b" (Fig. 5), two 200 mm long straight sections:

$$N = \frac{0.5 \times [18,300 - (200 + 200)]}{250 + 1.14 \times 14.5 - 0.43 \times 2.5} = 33.72 \approx 34$$

These 34 pitches will be distributed as 9 pitches in the outer parts and 16 in the center in this example. The average pitch in the outer parts, s_1 , will thus be:

$$s_1 = \frac{675}{9} = 75 \text{ mm (3.0 in)}$$

and the average pitch in the center, s_2 , will be:

$$s_2 = \frac{1,350}{16} = 84 \text{ mm (3.3 in)}$$

Check that $s \geq 4r + 2t$:

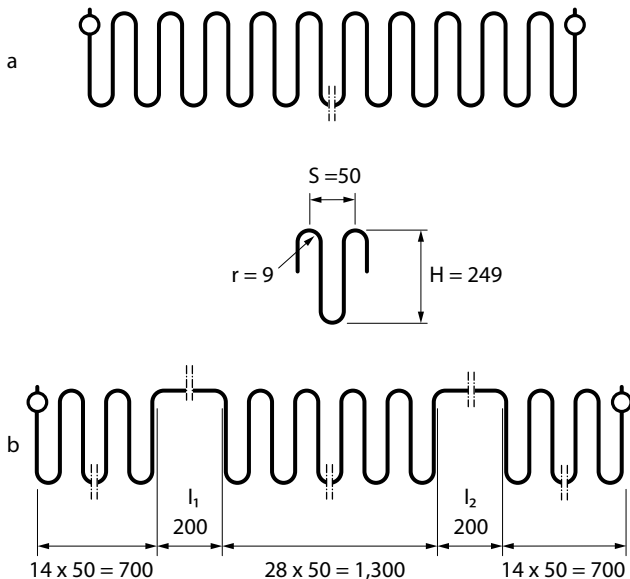
$$4r + 2t = 4 \times 14.5 + 2 \times 2.5 = 63 \text{ mm (2.5 in)} \rightarrow \text{ok}$$

CALCULATION OF CORRUGATION HEIGHT

The element height, or corrugation height, is calculated as:

$$H = \frac{0.5[\ell - (\ell_1 + \ell_2 + \dots + \ell_n)]}{N} - 1.14r + 0.43t$$

FIGURE 6



Element type "a" (Fig. 6), no straight sections:

$$H = \frac{0.5 \times (18,300)}{35} - 1.14 \times 14.5 + 0.43 \times 2.5 = 246 \text{ mm (9.7 in)}$$

Element "b" (Fig. 6), two 200 mm long straight sections:

$$H = \frac{0.5 \times [18,300 - (200 + 200)]}{34} - 1.14 \times 14.5 + 0.43 \times 2.5 = 248 \text{ mm (9.8 in)}$$

Check maximum corrugation height according to the recommendation. In this case height was limited to a maximum of 250 mm to fit in the furnace with the desired clearance.

CALCULATION OF ELEMENT TEMPERATURE

Element temperature, T_e , in °C is calculated as:

$$T_e = -273 + \sqrt[4]{\frac{P_s}{\epsilon \delta \times 5.67 \times 10^{-12}} + (T_f + 273)^4}$$

where ϵ is the emissivity of the strip, δ is a form factor, and T_f is the furnace temperature. The emissivity, ϵ , is 0.70 for Kanthal® alloys, and 0.88 for Nikrothal® alloys.

The form factor, δ , is calculated as:

$$\delta = \frac{b + t + \frac{s}{2} - \sqrt{\frac{s^2}{4} + b^2}}{b + t}$$

For this calculation, use the smallest pitch, s , in the design, which will likely be the hottest position. In this case, the smallest pitch is 75 mm, representing the outer parts of the bottom row (element type "b").

$$\delta = \frac{30 + 2.5 + \frac{75}{2} - \sqrt{\frac{75^2}{4} + 30^2}}{30 + 2.5} = 0.676$$

The calculated element temperatures will thus be:

$$T_e = -273 + \sqrt[4]{\frac{2.77}{0.70 \times 0.676 \times 5.67 \times 10^{-12}} + (1,100 + 273)^4} = 1,190 \text{ }^\circ\text{C (2,170 }^\circ\text{F)}$$

which is acceptable for Kanthal® AF.

COMPARISON BETWEEN NIKROTHAL® 80 AND KANTHAL® AF

The comparison has been made under the following assumed conditions: The same power, hot resistance, and cross sections for terminal and strip.

$$p_s = \frac{I^2 C_t}{\eta} = 303^2 \times \frac{1.07}{44,700} = 2.2 \frac{W}{cm^2} \left(14.2 \frac{W}{in^2} \right)$$

where η and C_t are looked up in the tables. C_t was looked up for 1,200 °C, i.e. 100 °C higher than the furnace temperature to take into account that the element is always hotter. The resistance per unit length, R_ℓ , used below for calculation of length, can also be found in the tables for the chosen strip dimension of Nikrothal® 80.

$$R_u = 2 \times \frac{4\ell_u \rho}{\pi d^2} = 2 \times \frac{4 \times 0.35 \times 1.09}{3.14 \times 20^2} = 0.0024 \Omega$$

$$R_{20} = \frac{R_T}{C_t} - R_u = \frac{0.363}{1.07} - 0.0024 = 0.0337 \Omega$$

$$\ell = \frac{R_{20}}{R_{20/m}} = \frac{0.337}{0.0145} = 23.2 \text{ m (76.1 ft)}$$

The density of Nikrothal® 80 is 8.30 g/cm³ (0.30 lb/in³), which gives a strip weight of 14.4 kg (31.8 lb).

For elements of type "a":

$$N = \frac{0.5 \times 23,200}{250 + 1.14 \times 14.5 - 0.43 \times 2.5} = 43.70 \approx 44$$

$$s = \frac{3,100}{44} = 70 \text{ mm (2.8 in)}$$

$$H = 0.5 \times \frac{23,200}{44} - 1.14 \times 14.5 + 0.43 \times 2.5 = 248 \text{ mm (9.8 in)}$$

For elements of type "b":

$$N = \frac{0.5 \times [23,200 - (200 + 200)]}{250 + 1.14 \times 14.5 - 0.43 \times 2.5} = 42.95 \approx 43$$

Distributed as 11 pitches per outer side, and 21 in the center region:

$$s_1 = \frac{675}{11} = 61.4 \text{ mm (2.4 in)}; s_2 = \frac{1,350}{21} = 64 \text{ mm (2.5 in)}$$

$$H = 0.5 \times \frac{[23,200 - (200 + 200)]}{43} - 1.14 \times 14.5 + 0.43 \times 2.5 = 250 \text{ mm (9.8 in)}$$

Calculation of element temperature, $s = 61.4 \text{ mm (2.4 in)}$:

$$\delta = \frac{30 + 2.5 + \frac{61.4}{2} - \sqrt{\frac{61.4^2}{4} + 30^2}}{30 + 2.5} = 0.624$$

The emissivity, ϵ , for Nikrothal® alloys is 0.88.

$$T_e = -273 + \sqrt[4]{\frac{2.2}{0.88 \times 0.624 \times 5.67 \times 10^{-12}} + (1,100 + 273)^4} = 1,164 \text{ °C (2,130 °F)}$$

ELEMENT "A" - COMPARISON TABLE STRIP 30 X 2.5 MM (1.2 X 0.1 IN)

	KANTHAL® AF	NIKROTHAL®
Power, kW	33.33	33.33
Hot resistance, Ω	0.363	0.363
Element height, mm (in)	246 (9.7)	250 (9.8)
Strip length, m (ft)	18.3 (61.9)	23.2 (76.1)
Strip weight, kg (lbs)	9.8 (21.6)	14.4 (31.8)
Number of supports	34	43

The strip length is 21% shorter and the element weight is 32% less with Kanthal® AF compared with Nikrothal® 80. The number of supporting pins is reduced by 21% with Kanthal® AF.